

- / -

2

113.27

137

%17

$(3.82 \cdot 10^6 \text{N/m}^2)$

.(30Kw)

TIRE RECYCLING AS A TOOL FOR MANUFACTURING RUBBER COUPLER

Dr. Tahseen A. Hosain
Technical college / Najaf

Hassan H. Salman
Technical college / Najaf

ABSTRACT

The paper aimed to use the old rubber tires which causing a big problem to the environment and its need much money and efforts to disposing it safely. In Iraq there is more than 2 million used tires disposed to the environment yearly whereas in the US there is 2 billion one.

One of the tyre recycling methods , is the use of tyre plies to produce a new rubber parts used in the engineering and industrial purposes as the bridge and machine dampers , this trend of recycling doesn't take a sufficient care comparing with the other methods .

In the current research the study focusing on the mechanical properties of the used tyres and comparing them with the new one , and suggesting a new method to use the recycled tyres as an engineering parts (for example the coupler lied between the IC engine and the generating head in the generators .

One of the results obtained from the experiments, there is no significant difference between the mechanical properties of the old and new tyres , (in the tensile test – the breaking force of the tensile specimen of the new tyre is 137 N and for the old one is 113.27 N , but they are same in the elongation) . A dimensions of a flexible coupling suggested as a computational example and when using the experimental value of shear stress for the cord-rubber composite lamina ($3.82 \cdot 10^6 \text{N/m}^2$) the calculation showing that the number of strips for the coupler should be two with 2 ply lamina or 4 with one ply lamina at engine power less than 30 Kw .

KEYWORDS: Tyre recycling , Rubber coupling , Rubber properties

:

	E_1
	E_2
معامل بايسون (poisson's ratio) في المستوي (2-1) .	ν_{12}
(2-1) .	G_{12}
	D
	t
	E_c
	E_r
	V_c
	V_r
	G_c
	G_r
	e
	f
	τ
	T
	A_L
	n,n'

750

50000

32

1983

Tire recycling

-1

(.....)

-2

(online .org.uk, 2007www.wast).

-3

()

()

(14) ()
 (32)

1.6 (Wear indicator)
 12

1983

-: (Tread) .1

(Rolling resistance) (wet grip)
 (heat accumulation) (wear)

-: (Breaker bandage) .2

-: (Ply topping) .3

-: (Steel cord topping breakers) .4

-: (Side wall) .5

(Bead) (Tread)

-: (Bead wire coating) .6

(Ring)

.(Wheel)

-: (Bead apex) .7

-: (Inner liner) .8

]

[Parkash , 2007

1

1 ()

-:

(Non – destructive) :

-:

: (Tyreregroving) -1

: (Tyre remolding) -2

: (Using tyres in seaport) -3

: -4

(Destructive) :

-: *

: (grinding) -1

(80-) : (cryogenic) -2

: (de-vulcanization) -3

(reclaimmeter)

: (Advance Molecular Agitation Technology - AMAT) -4

2000

: (Ply separation) -5

-:

: (Rubber washers) -أ

. (Plate and nut)

: (Rubber belts) -ب

: (Rubber containers) -ت

.....

: (Machine vibration isolation or Damper) -ث

www. wast online .org. uk ,2007 .

60% : (Energy recovery) -6

:

500-)

: (pyrolysis process)

-1

(900

()

[Paul , 1999.]

(tire – derived fuel)

-ب

300%

125%

()

:

2

. E_1 (young modulus) -1

. E_2 -2

. ν_{12} (1 -2) (Poisson 's ratio) -3

. G_{12} (1 -2) (shear modulus) -4

$$E_1 = E_c V_c + E_r V_r \tag{1}$$

$$V_c + V_r = 1 \quad , \quad V_c = \frac{\pi D^2 e}{4t}$$

(100-1000)

:

$$E_1 = E_c V_c \tag{2}$$

1-2 (12v)

$$v_{12} = V_v + V_v \tag{3}$$

(transvers direction)

$$\frac{1}{E^2} = \frac{V_c}{E_c} + \frac{V_r}{E_r} \tag{4}$$

: >> E_r E_c

$$E_2 = \frac{E_r}{V_r} \tag{5}$$

G₁₂ (shear modulus)

$$G_{12} = \frac{G_r}{V_r} \tag{6}$$

$$G_c \gg G_r$$

Asaad , 2006 . (s.klark, Akasaka-Hirano,Gough-Tangorra)

Westerberg and Macsik,2001; ASTM,1998 ; Drescher and Newcomb)

1994; Edil and Bosscher, 1992; Humphrey, et al., 1993 and Huphrey and Manion, 1992)

(shreds)

.6

· φ ()

Wu, Benda and Cauley,) (triaxial tests)

(1997 and Benda, 1995

(Humphrey , et.al 1993) 12% ()

7

.(machine foundation)

.3

3

°4

Baker,2003.

: (Tyre coupler)

(3) .() -1

.() -2

IRH 70 ± 5° () -3

(5500rpm 900 rpm 12500 Kw 15 Kw)

Gitin , 1995.

flexible coupling

4

pressure disc

$$(t) \quad (w)x = \quad)$$

:

(Generator) (IC engine) :

4 10 hr/day rpm3000 30 Kw
 . °(30-50)

Nominal power = 30 Kw

Design power = Nominal power * f₁ * f₂ * f₃

f₁ = minimum factor of safety for types of duty .

f₂ = factor of safety for daily operation .

f₃ = factor of safety for frequency of start .

from tables : (f₁ = 1 , f₂ = 1.12 , f₃ = 1.09)

Then :

Design power = 30 * 1 * 1.12 * 1.09 = 37 Kw .

Power in Kw / speed in rpm Ratio PKw/rpm .

Is

$$\frac{37}{3000} = \frac{P.Kw}{rpm}$$

$$= 0.01233 \text{ Kw/rpm.}$$

Kw / 100rpm = 1.233 Kw / 100rpm .

From tables : The Dimensions suitable for the coupling have Kw / 100rpm. > 1 and < 2.1 are as follow .

Torque N.m = 150 N.m

Mass speed = 3200 rpm

Bore in minimum rough= 30 mm

Bore in maximum rough = 55 mm

A = 212 mm , B = 95 mm , C = 169 mm , D = 157 mm ,

E = 83 mm , F = 56 mm , G = 45 mm .

. 3

(A , B , C , D , E , F , G)

:

(0.1 x 0.002)

$$\tau = \frac{T}{0.5A * A_L} = \frac{150}{0.106} = 7.07 * 10^6 \text{ N/m}^2$$

$$.3.82 * 10^6 \text{ N/m}^2$$

saad.2006

G

:

$$\dot{n} = \frac{\tau}{G} = \frac{7.07 * 10^6}{3.82 * 10^6} = 1.845 \approx 2$$

: (f = 2)

$$n = f * \dot{n} = 2 * 2 = 4$$

.

()

:

-1

.

113.27

(breaking force)

-2

. 5

%17

137

-3

-4

4°

3

.

30 Kw

.

1 1

.1

. 1983

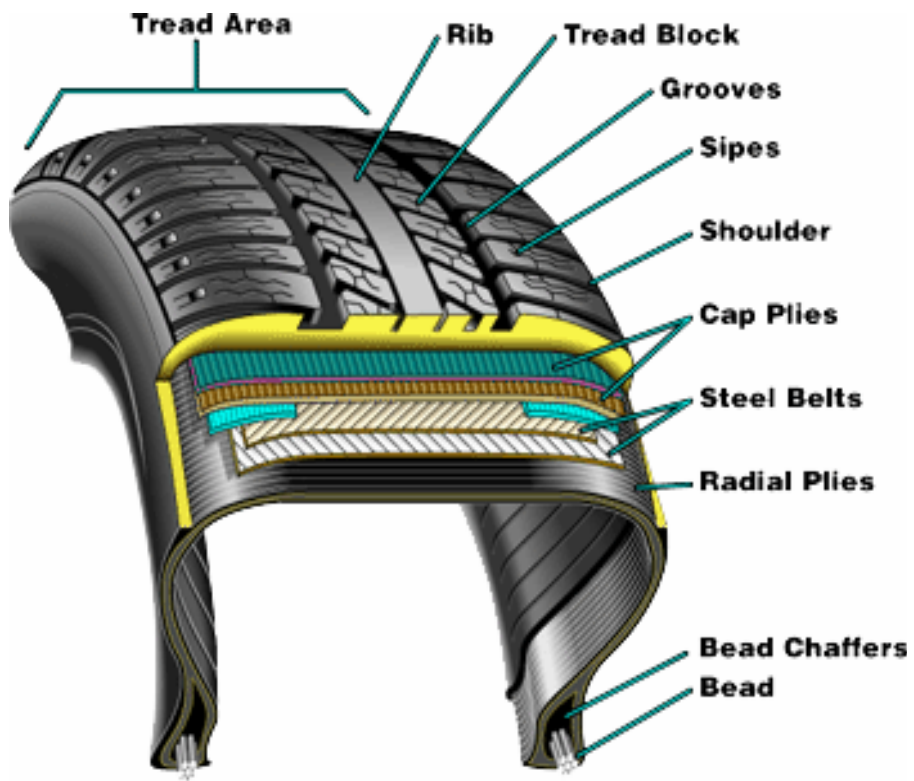
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1

		%
	100	51
	50	26
	25	13
	2	1
	4	2
*	15	7

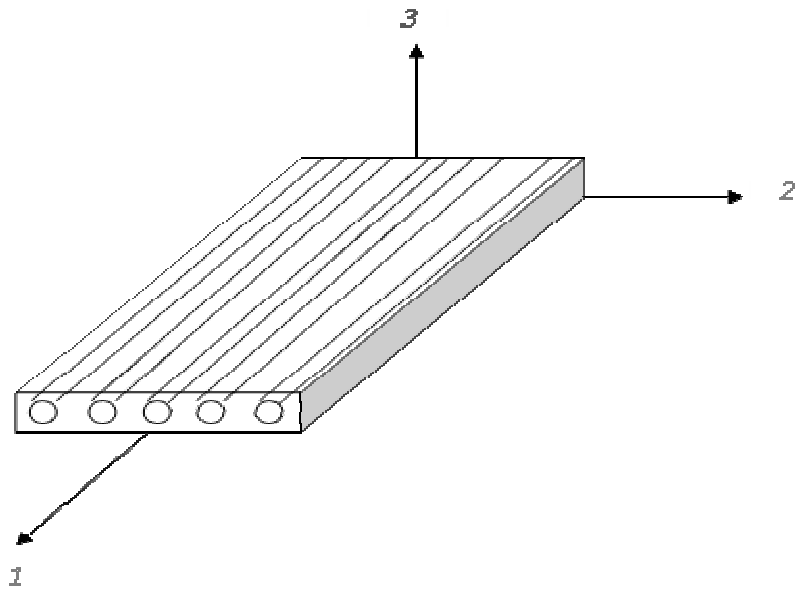
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(www.wastonline.org.uk)

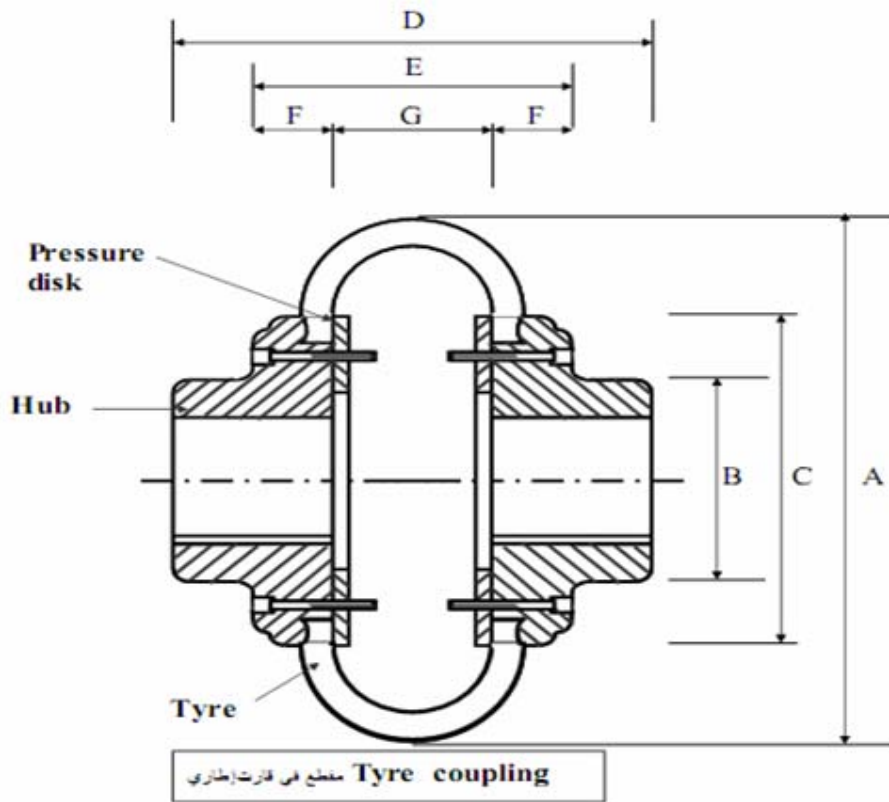


(Parkash,2007)

1

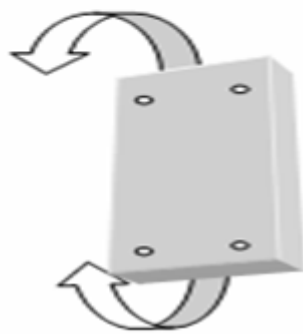
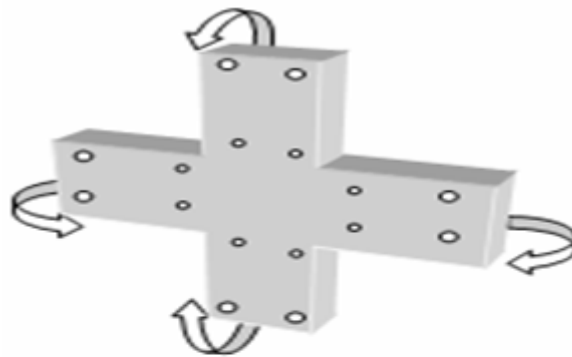


2

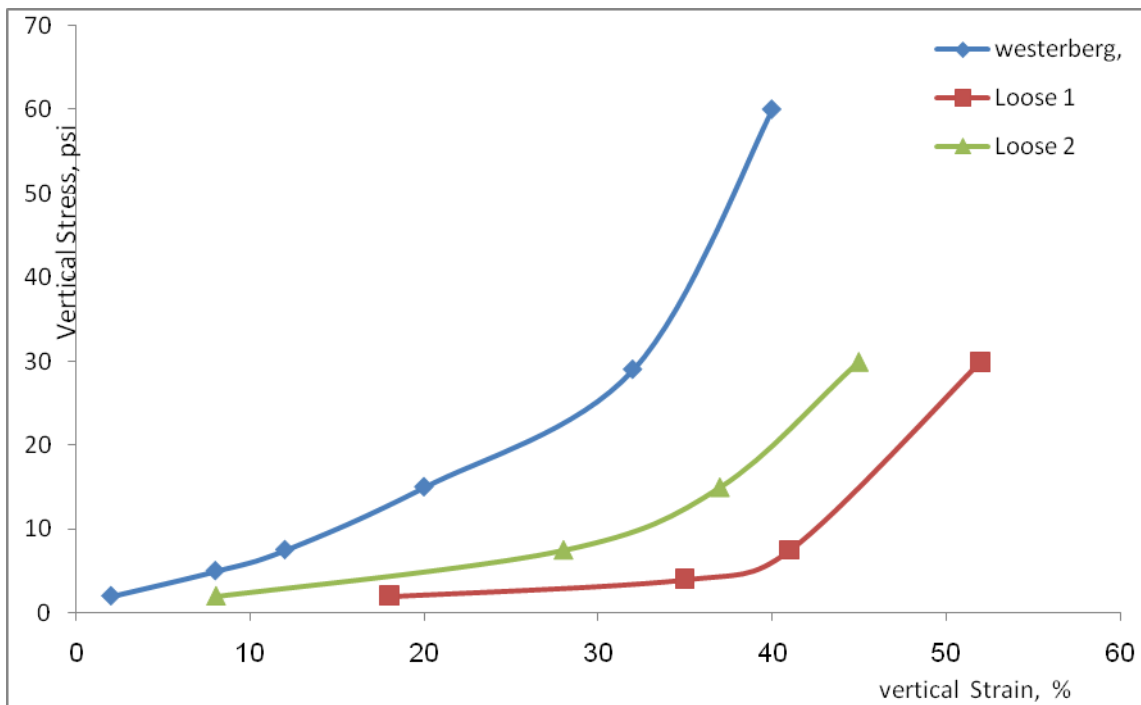
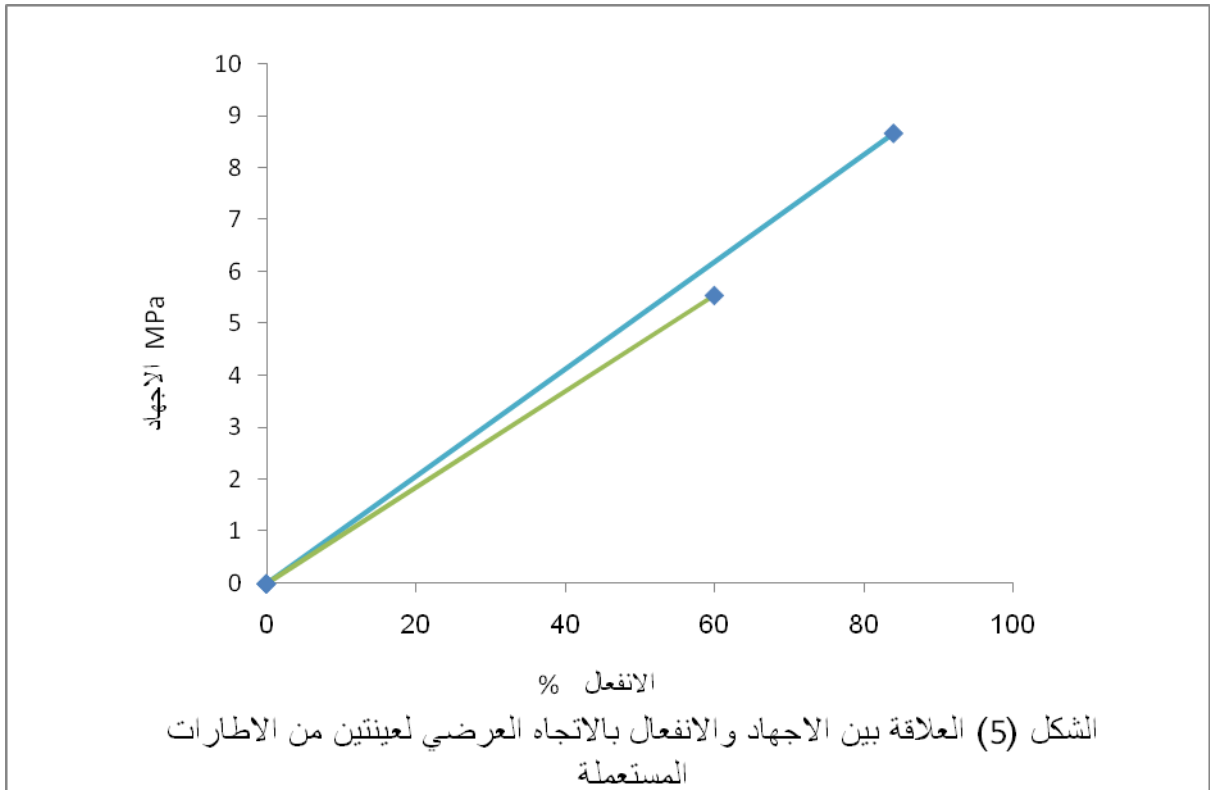


(Gitin,1995)

3

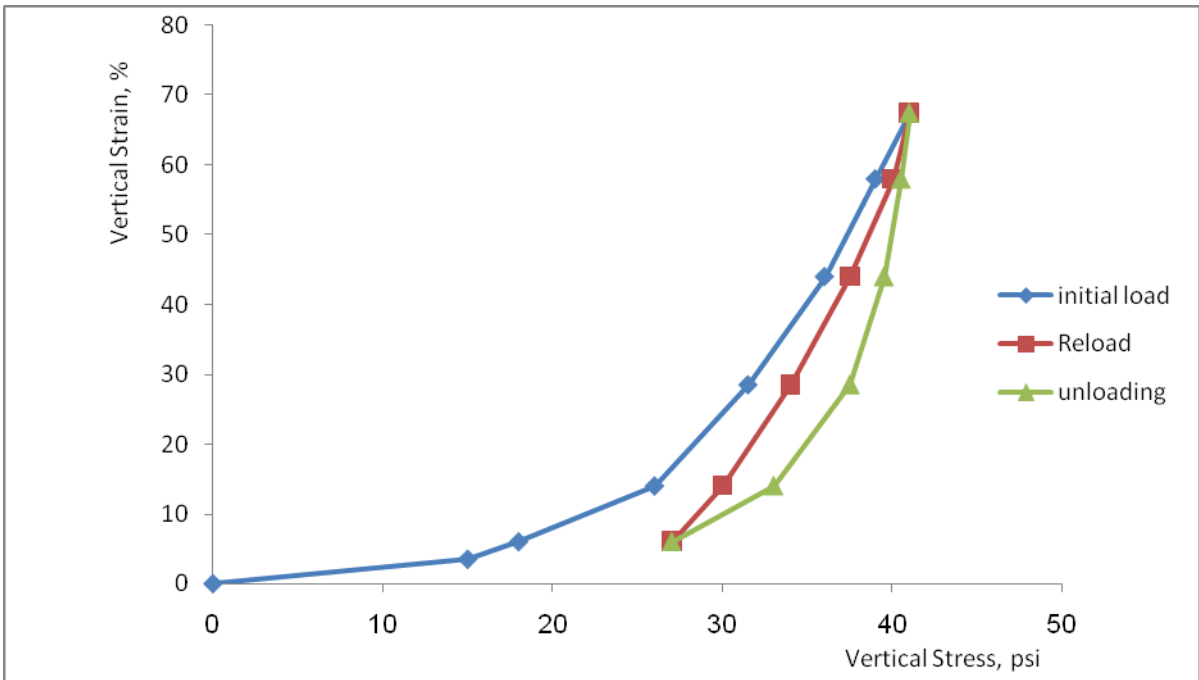


4



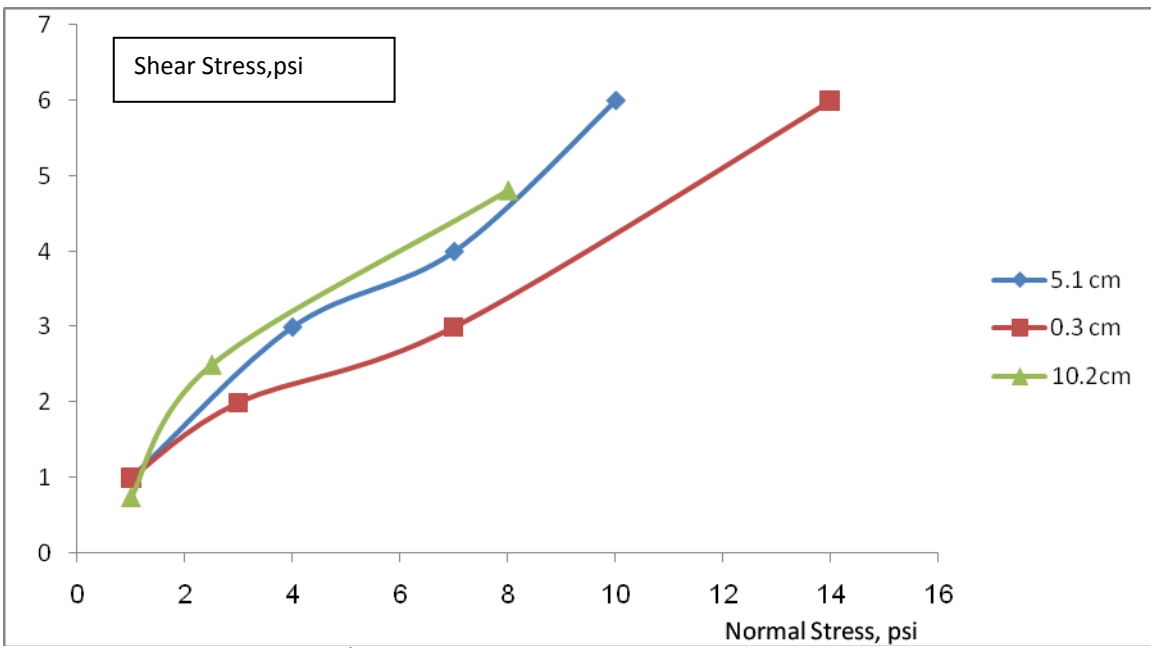
(Baker,2003)

6



7

(Humphrey, 1993)



الشكل رقم 8 إجهاد القص لشرائح الإطارات المستعملة محسوبة على أساس اختبار القص المباشر (Baker,2003)